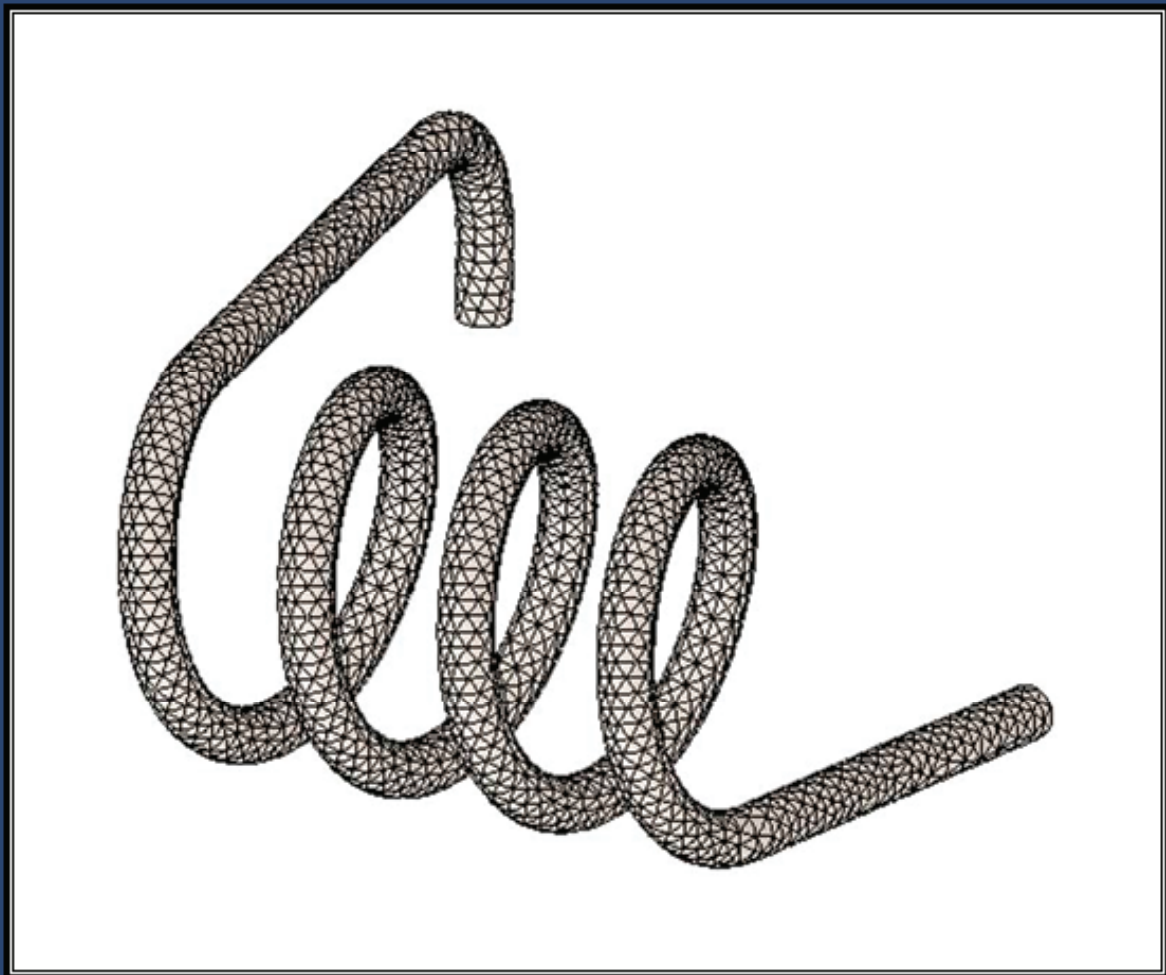


Engineering Analysis with SolidWorks Simulation 2010



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2: Static analysis of a plate

Topics covered

- ❑ Using SolidWorks **Simulation** interface
- ❑ Linear static analysis with solid elements
- ❑ Finding reaction forces
- ❑ Controlling discretization errors by the convergence process
- ❑ Finding reaction forces
- ❑ Presenting FEA results in desired format

Project description

A steel plate is supported and loaded, as shown in Figure 2-1. We assume that the support is rigid (this is also called built-in support or fixed support) and that a 100000N tensile load is uniformly distributed along the end face, opposite to the supported face.

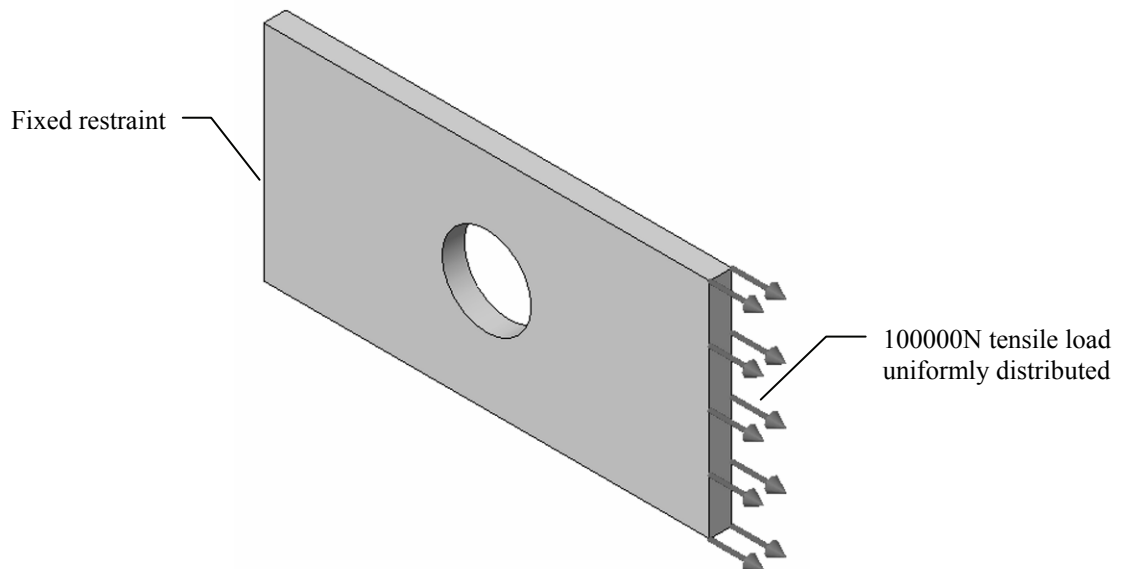


Figure 2-1: SolidWorks model of a rectangular plate with a hole

We will perform displacement and stress analysis using meshes with different element sizes. Note that repetitive analysis with different meshes does *not* represent standard practice in FEA. The process does however produce results which are useful in gaining more insight into how FEA works.

Procedure

In SolidWorks, open the model file called HOLLOW PLATE. Verify that SolidWorks **Simulation** is selected in the **Add-Ins** list (Figure 2-2).

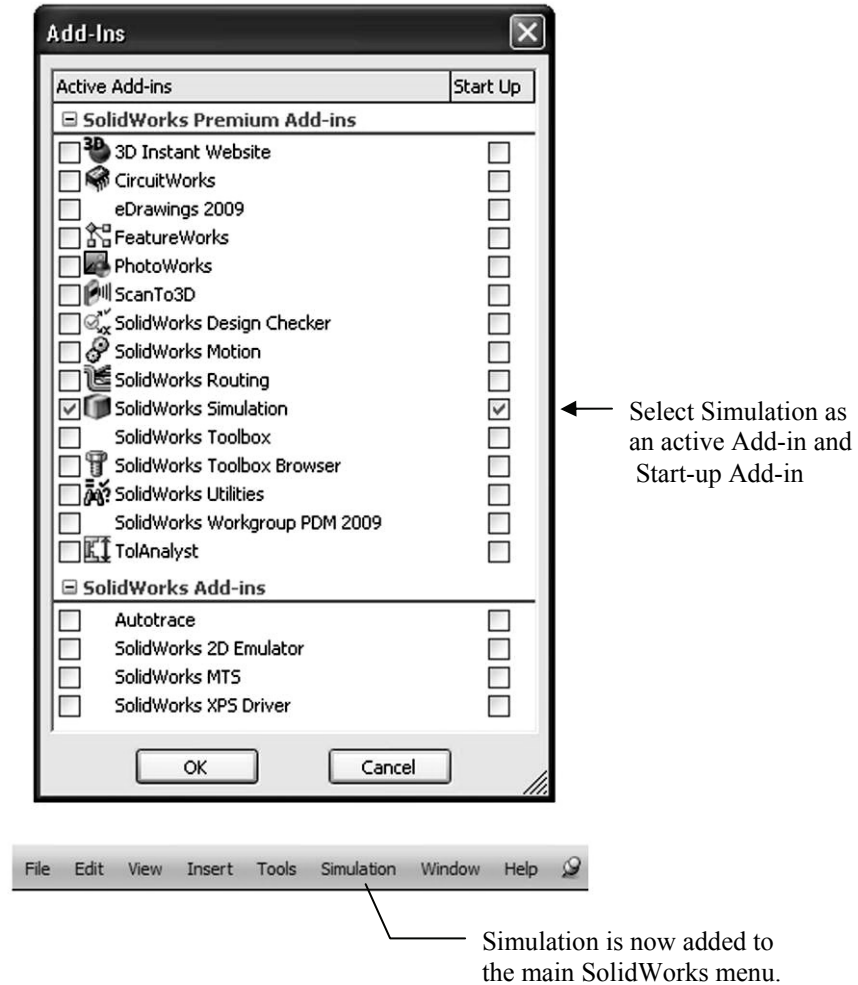


Figure 2-2: Add-Ins list and SolidWorks Simulation Manager tab

Verify that SolidWorks Simulation is selected in the list of Add-Ins (top). Once Solid Works Simulation has been added, Simulation shows in the main SolidWorks tool menu (bottom).

It is necessary to add the **Simulation** tab to the **Command Manager** if it is not visible. If the **Simulation** tab has been added but is still not showing, follow steps explained in Figure 2-3.

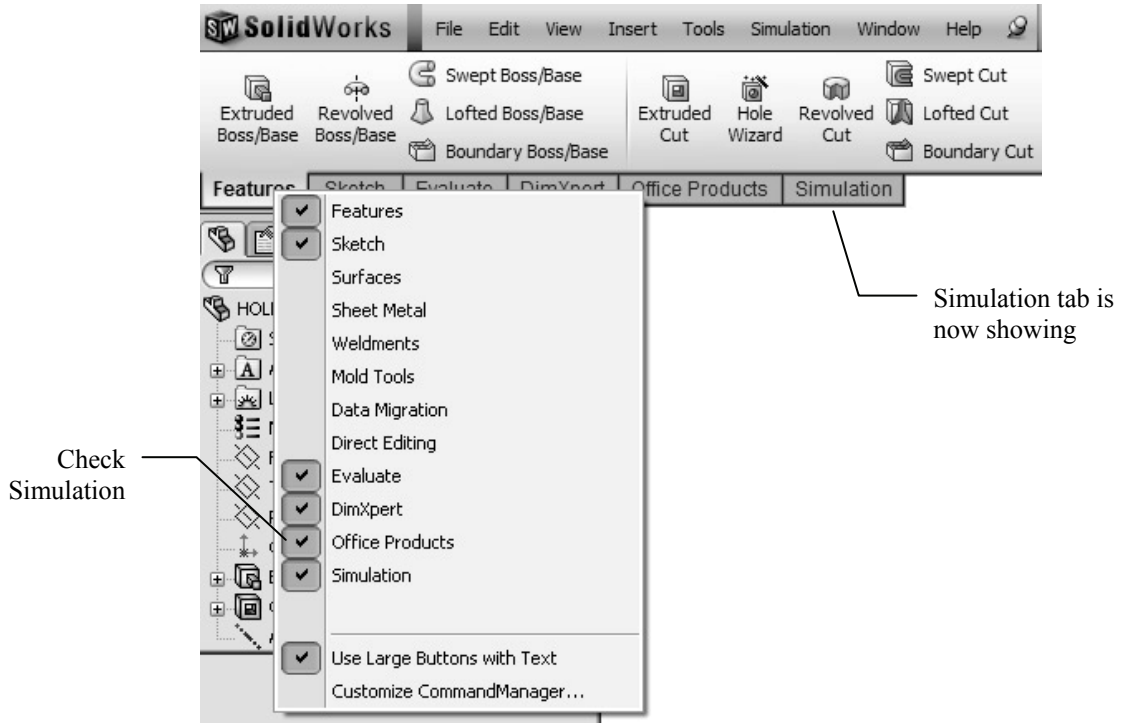


Figure 2-3: How to display Simulation tab in SolidWorks Command Manager

Right-click any tab in Command Manager and check “Simulation” from the pop-up menu to make the Simulation visible.

Before we create the FEA model, let's review the **Simulation** main menu along with its **Options** window (Figure 2-4).

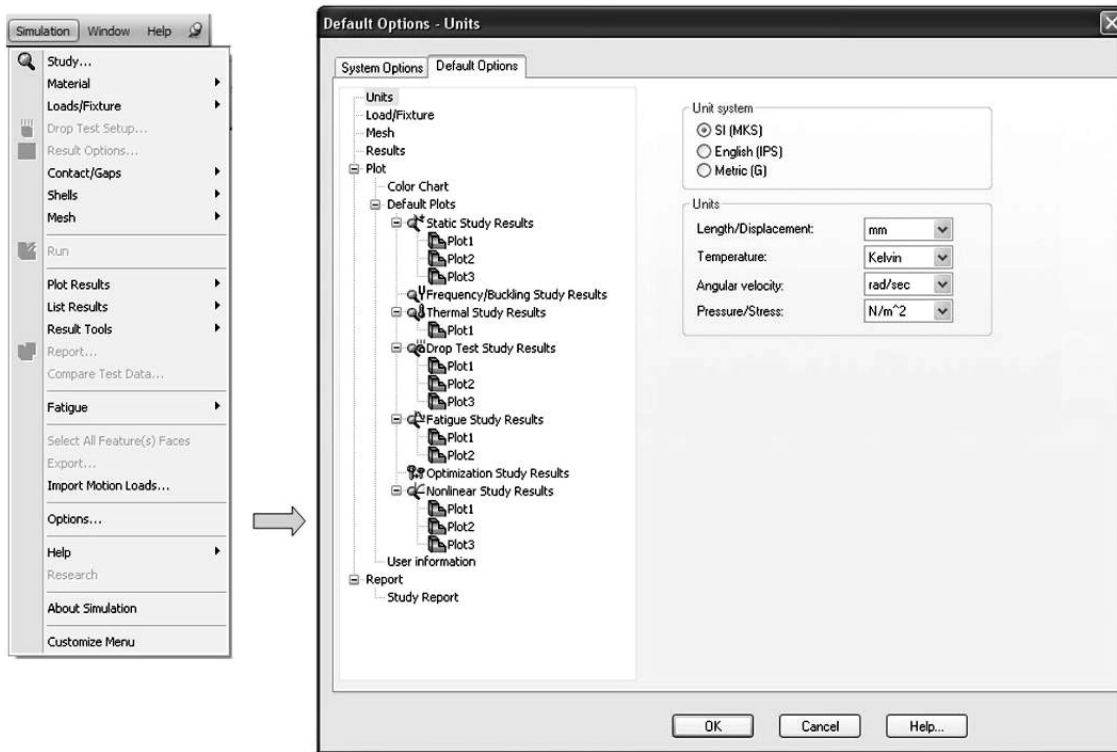


Figure 2-4: Simulation main menu (left) and Options window; shown is Default Options tab

The Default Options window has two tabs. Review both tabs before proceeding with the exercise. Note that Default Plots can be added, deleted or grouped into sub-folders which are created by right-clicking on the Static Study Results folder, Thermal Study Results folder etc. As shown above, we use the SI system as specified in Default Options tab.

Creation of an FEA model starts with the definition of a study. To define a new study, select **New Study** either from the **Simulation** menu (Figure 2-5 left) or the **Simulation** Command Manager (Figure 2-5 right). Name the study *tensile load 01*.

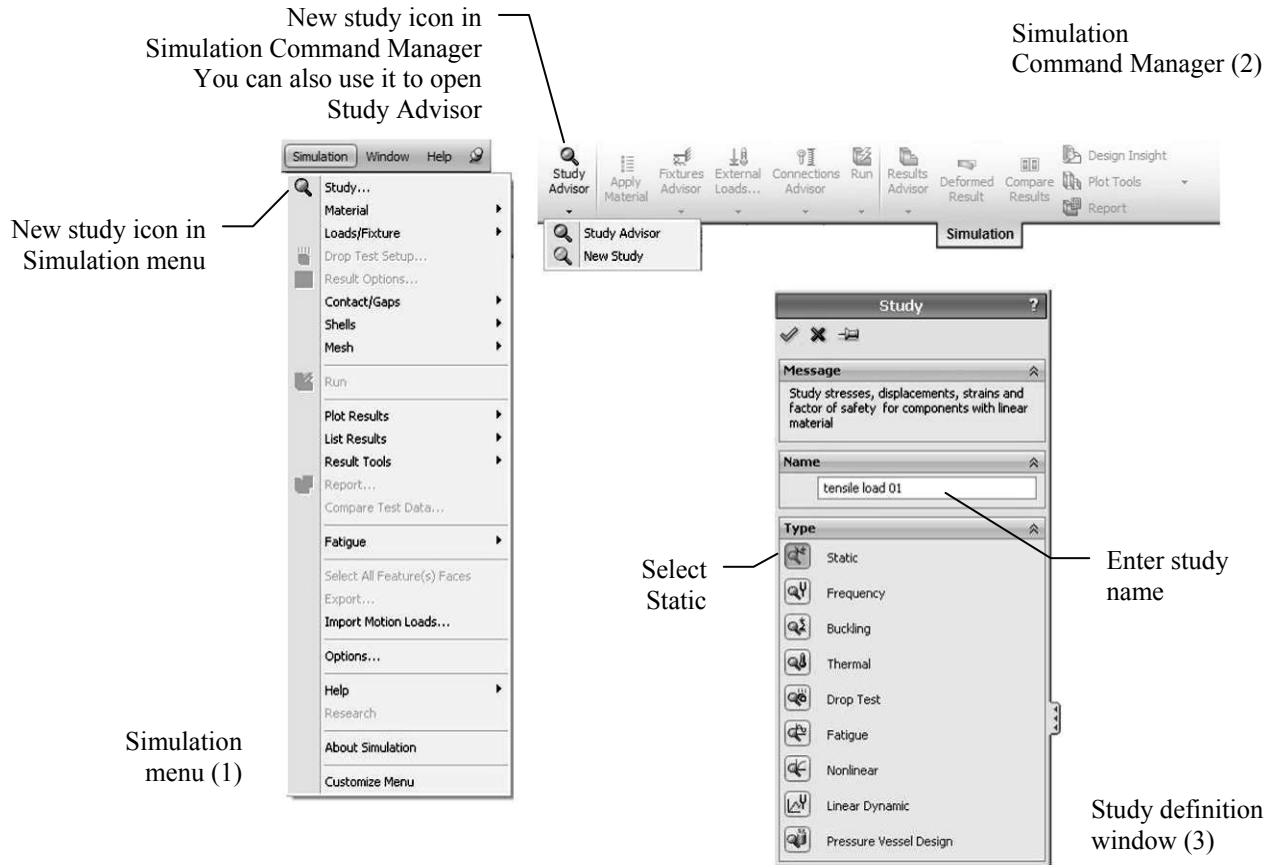


Figure 2-5: Creating a New Study

A study can be created either using Simulation menu (1) or Simulation Command Manager (2) Also shown is Study definition window (3)

Once a new study has been created, all Simulation Commands can be invoked in three ways:

- ☐ From the Simulation main menu
- ☐ From the Simulation tab in Command Manager
- ☐ By right-clicking appropriate items in the study window

In this book, we will most often use the third method.

When a study is defined, SolidWorks **Simulation** creates a study window located below the SolidWorks **Feature Manager** window and places several folders in it. It also adds a study tab that provides access to the window (Figure 2-6).

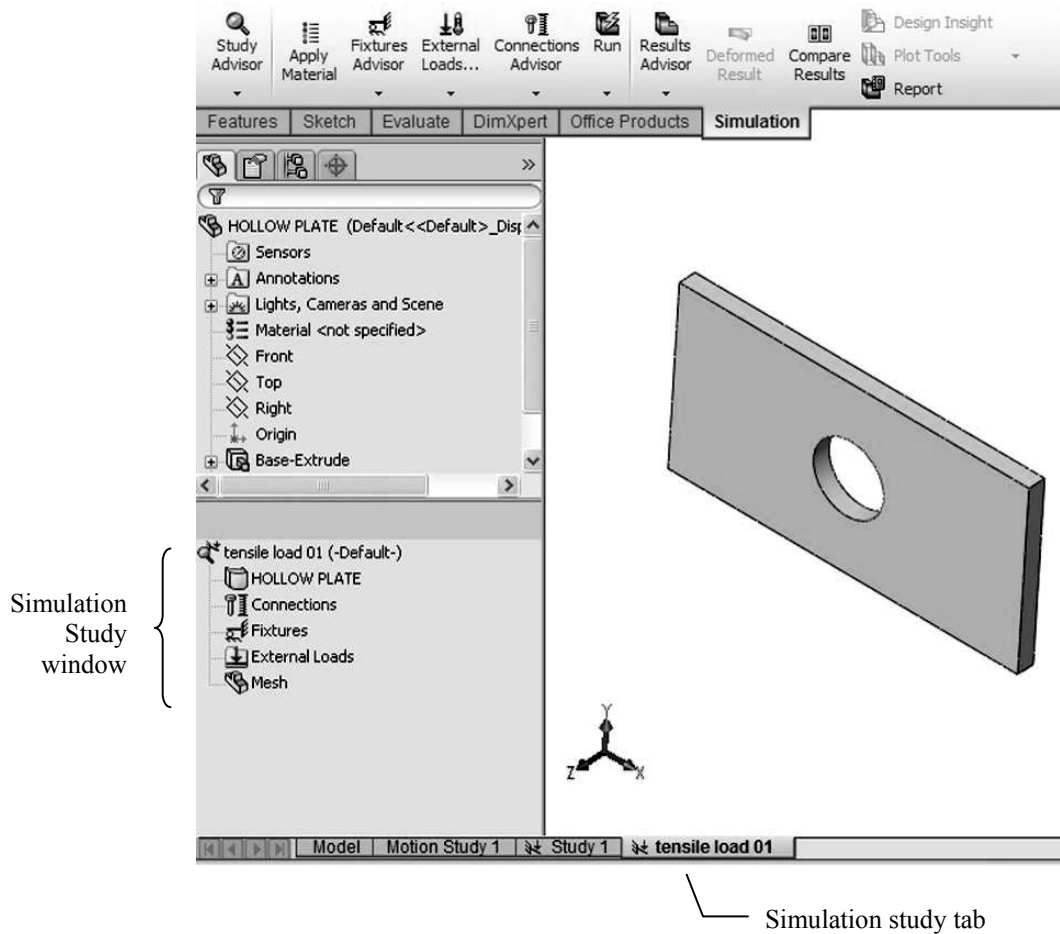


Figure 2-6: Simulation window and Simulation tab

You can switch between SolidWorks Model, Motion Studies and Simulation Studies by selecting the appropriate the tab.

We are now ready to define the analysis model. This process generally consists of the following steps:

- ❑ CAD geometry idealization and/or simplification in preparation for analysis. This is usually done by creating analysis specific configuration and making your changes there
- ❑ Material properties assignment
- ❑ Restraints application
- ❑ Load application

In this case, the geometry does not need any preparation because it is already very simple, therefore we can start by assigning material properties.

Notice that if a material is defined for a **SolidWorks** part model, material definition is automatically transferred to the **Simulation** model. Assigning a material to the SolidWorks model is actually a preferred modeling technique, especially when working with an assembly consisting of parts with many different materials. We will do this in later exercises.

To apply material to the **Simulation** model, right-click the HOLLOW PLATE folder in the *tensile load 01* simulation study and select **Apply/Edit Material** from the pop-up menu (Figure 2-7).

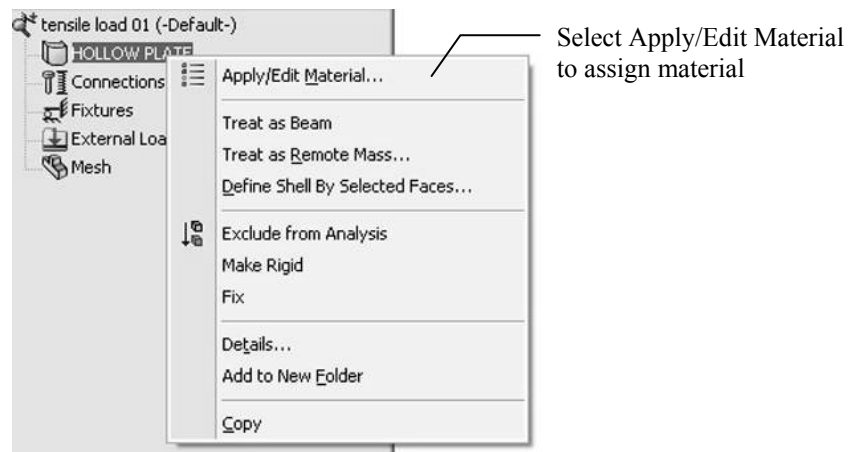


Figure 2-7: Assigning material properties

The action in Figure 2-7 opens the **Material** window shown in Figure 2-8.

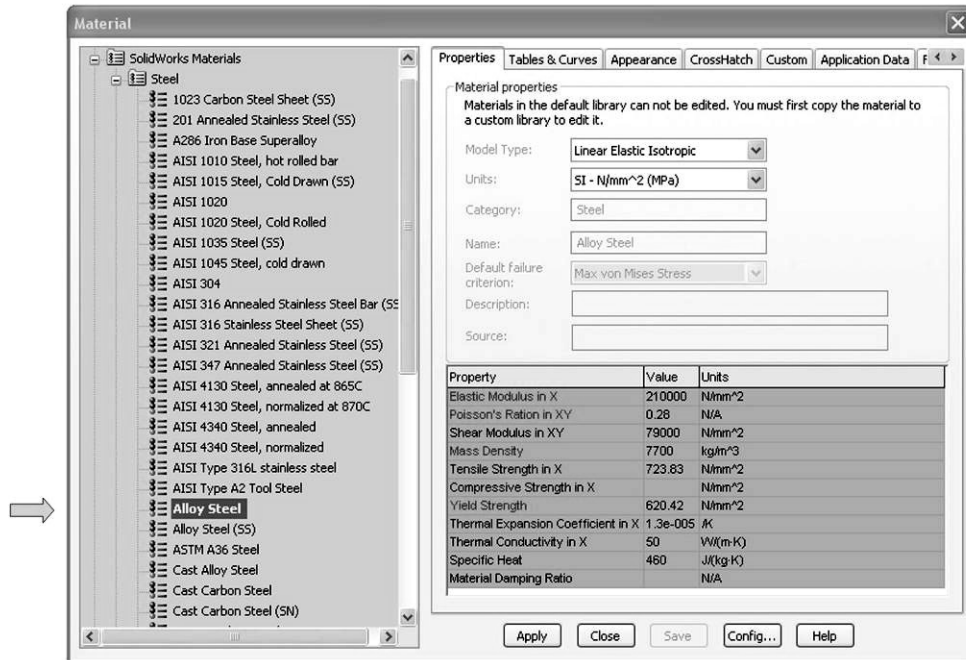


Figure 2-8: Material window

Select Alloy Steel to be assigned to the model.

In the **Material** window, the properties are highlighted to indicate the mandatory and optional properties. A red description (Elastic modulus, Poisson's ratio) indicates a property that is mandatory based on the active study type and the material model. A blue description (Mass density, Tensile strength, Compressive strength, Yield strength, Thermal expansion coefficient) indicates optional properties. A black description (Thermal conductivity, Specific heat, Material damping ratio) indicates properties not applicable to the current study.

In the **Material** window select **From library files** in the **Select material source** area, and then select **Alloy Steel**. Select **SI** units under the **Properties** tab (other units could be used as well). Notice that the *HOLLOW PLATE* folder in *tensile load 01* study now shows a check mark and the name of the selected material to indicate that a material has been assigned. If needed, you can define your own material by selecting **Custom Defined** material.

Material definition consists of two steps:

- ☐ Material selection (or material definition if a custom material is used)
- ☐ Material assignment (either to all solids in the model, selected bodies of a multi-body part, or to selected components of an assembly)

Having assigned the material, we now move to defining the loads and restraints. To display the pop-up menu that lists the options available for defining restraints, right-click the *Fixtures* folder in the *tensile load 01* study (Figure 2-9).

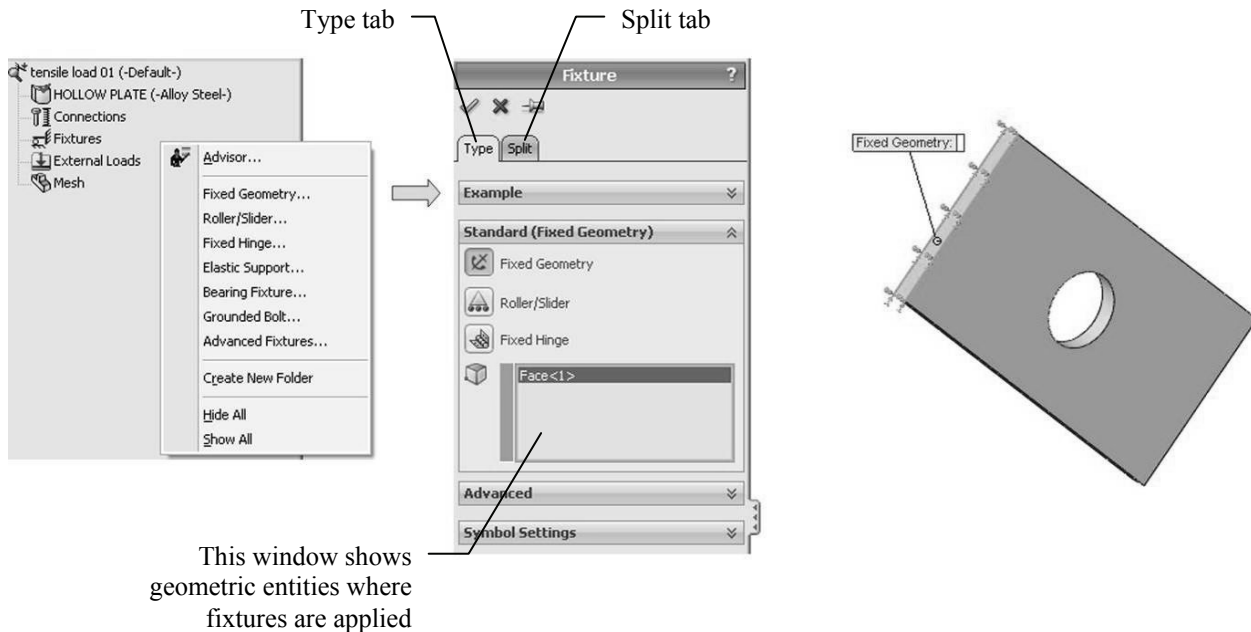


Figure 2-9: Pop-up menu for the Fixtures folder and Fixture definition window

All restraints definitions are done in the Type tab. The Split tab is used to define a split face where a restraint is to be defined. The same can be done in SolidWorks by defining a Split Line.

Once the **Fixtures** definition window is open, select the **Fixed Geometry** restraint type. Select the end-face entity where the restraint is applied.

Note that in SolidWorks **Simulation**, the term “Fixture” implies that the model is firmly “fixed” to ground. However, aside from **Fixed Geometry**, which we have just used, all other types of fixtures restrain the model in certain directions while allowing movements in other directions. Therefore, the term “restraint” may better describe what happens when choices in the **Fixture** window are made. In this book we will switch between terms “fixture” and “restraint” freely.

Before proceeding, explore other types of restraints accessible through the **Fixture** window. All types of restraints are divided in two groups: **Standard** and **Advanced**. Review animated examples available in the **Fixture** window and study the following chart.

Standard Fixtures	
Fixed	Also called built-in or rigid support, all translational and all rotational degrees of freedom are restrained.
Immovable (No translations)	Only translational degrees of freedom are constrained, while rotational degrees of freedom remain unconstrained. If solid elements are used (like in this exercise), Fixed and Immovable restraints would have the same effect because solid elements do not have rotational degrees of freedom. Therefore, Immovable restraint is not available if solid elements are used.
Hinge	Applies only to cylindrical face and specifies that the cylindrical face can only rotate about its own axis. This condition is identical to selecting the On cylindrical face restraint type and setting the radial and axial components to zero.
Advanced Fixtures	
Symmetry	Applies symmetry boundary conditions to a flat face. Translation in the direction normal to the face is restrained and rotations about the axes aligned with the face are restrained.
Roller/Sliding	Specifies that a planar face can move freely on its plane but not in the direction normal to its plane. The face can shrink or expand under loading.
Use reference geometry	Restraints a face, edge, or vertex only in certain directions, while leaving the other directions free to move. You can specify the desired directions of restraint in relation to the selected reference plane or reference axis.
On flat face	Provides restraints in selected directions, which are defined by the three directions of the flat face where restraints are being applied.
On cylindrical face	This option is similar to On flat face , except that the three directions of a cylindrical face define the directions of restraints.
On spherical face	Similar to On flat face and On cylindrical face . The three directions of a spherical face define the directions of applied restraints.
Cyclic symmetry	Allows analysis of a model with circular patterns around an axis by modeling a representative segment. The segment can be a part or an assembly. The geometry, restraints, and loading conditions must be identical for all other segments making up the model. Turbine, fans, flywheels, and motor rotors can usually be analyzed using cyclic symmetry.

When a model is fully supported (as it is in our case), we say that the model does not have any rigid body motions (the term “rigid body modes” is also used), meaning it cannot move without experiencing deformation.

Note that the presence of restraints in the model is manifested by both the restraint symbols (showing on the restrained face) and by the automatically created icon, **Fixture-1**, in the *Fixtures* folder. The display of the restraint symbols can be turned on and off by either:

- ❑ Right-clicking the *Fixtures* folder and selecting **Hide All** or **Show All** in the pop-up menu shown in Figure 2-9 , or
- ❑ Right-clicking the fixture icon and selecting **Hide** or **Show** from the pop-up menu.

Now define the load by right-clicking the *External Loads* folder and selecting **Force** from the pop-up menu. This action opens the **Force** window as shown in Figure 2-10.

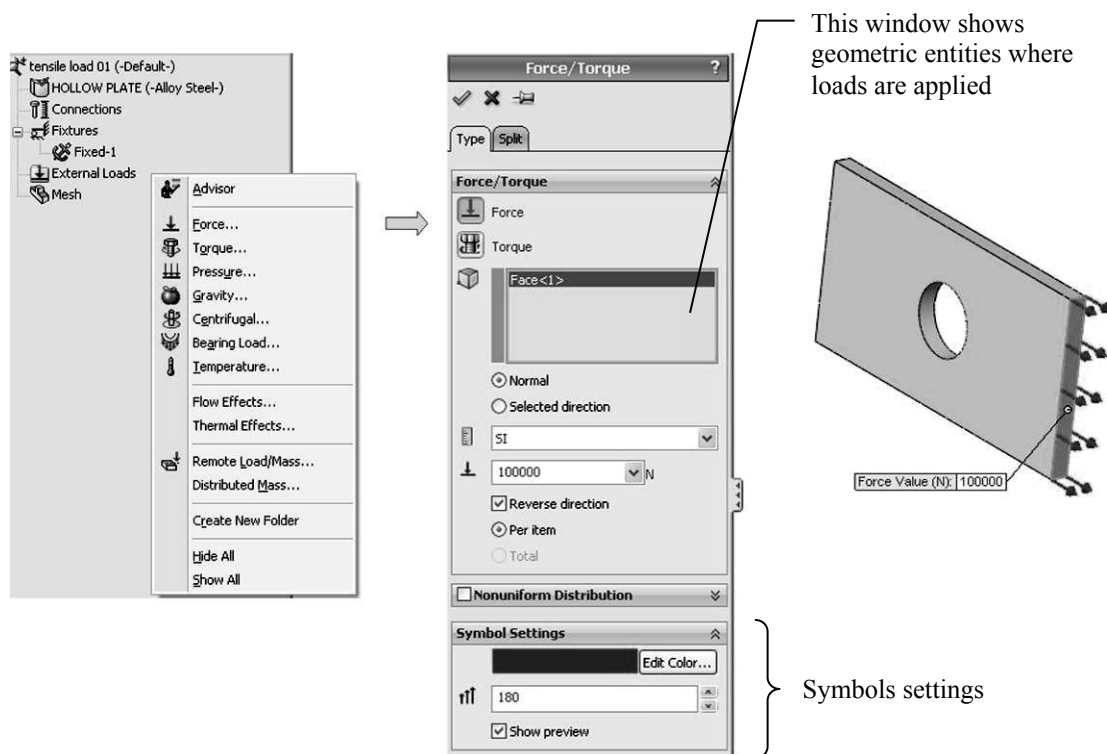


Figure 2-10: Force window

The Force window displays the selected face where the tensile force is applied. If only one entity is selected, there is no distinction between Per Item and Total. This illustration also shows the model with symbols of applied restraint and load. Load symbols have been enlarged by adjusting the Symbols Settings.

In the **Type** tab, select **Normal** in order to load the model with a 100000N tensile force uniformly distributed over the end face, as shown in Figure 2-10. Check the **Reverse direction** option to apply a tensile load.

Generally, forces can be applied to faces, edges, and vertices using different methods, which are reviewed below:

Force normal	Available for flat faces only, this option applies load in the direction normal to the selected face.
Force selected direction	<p>This option applies a force or a moment to a face, edge, or vertex in the direction defined by the selected reference geometry.</p> <p>Moments can be applied only if shell elements are used. Shell elements have six degrees of freedom per node: three translations and three rotations, and can take a moment load.</p> <p>Solid elements only have three degrees of freedom (translations) per node and, therefore, cannot take a moment load directly.</p> <p>If you need to apply moments to solid elements, they must be represented with appropriately applied forces.</p>
Torque	This option applies torque (expressed by traction forces) about a reference axis using the right-hand rule.

Try using the click-inside technique to rename the **Fixture-1** and **Force/Torque-1** icons. Note that renaming using the click-inside technique works on all icons in SolidWorks **Simulation**.

The model is now ready for meshing. Before creating a mesh, let's make a few observations about defining the geometry, material properties, loads and restraints.

Geometry preparation is a well-defined step with few uncertainties. Geometry that is simplified for analysis can be compared with the original CAD model.

Material properties are most often selected from the material library and do not account for local defects, surface conditions, etc. Therefore, definition of material properties usually has more uncertainties than geometry preparation.

The definition of loads is done in a few quick menu selections, but involves many assumptions. Factors such as load magnitude and distribution are often only approximately known and must be assumed. Therefore, significant idealization errors can be made when defining loads.

Defining restraints is where severe errors are most often made. For example, it is easy enough to apply a fixed restraint without giving too much thought to

the fact that a fixed restraint means a rigid support – a mathematical abstraction. A common error is over-constraining the model, which results in an overly stiff structure that underestimates displacements and stresses. The relative level of uncertainties in defining geometry, material, loads, and restraints is qualitatively shown in Figure 2-11.

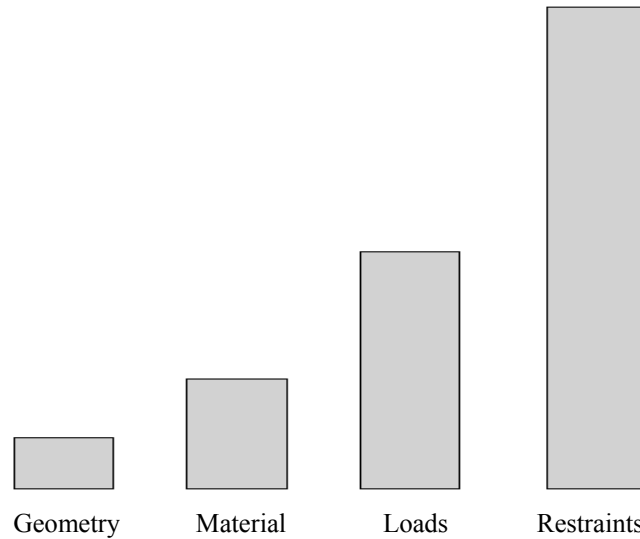


Figure 2-11: Qualitative comparison of uncertainty in defining geometry, material properties, loads, and restraints

The level of uncertainty (or the risk of error) has no relation to time required for each step, so the message in Figure 2-11 may be counterintuitive. In fact, preparing CAD geometry for FEA may take hours, while applying restraints and loads takes only a few clicks.

In all of the examples presented in this book, we assume that definitions of material properties, loads, and restraints represent an acceptable idealization of real conditions. However, we need to point out that it is the responsibility of the FEA user to determine if all those idealized assumptions made during the creation of the mathematical model are indeed acceptable.

Before meshing the model, we need to verify under the **Default Options'** **Mesh** tab that **High** mesh quality is selected. The **Options** window can be opened from SolidWorks **Simulation** menu as shown in Figure 2-12.

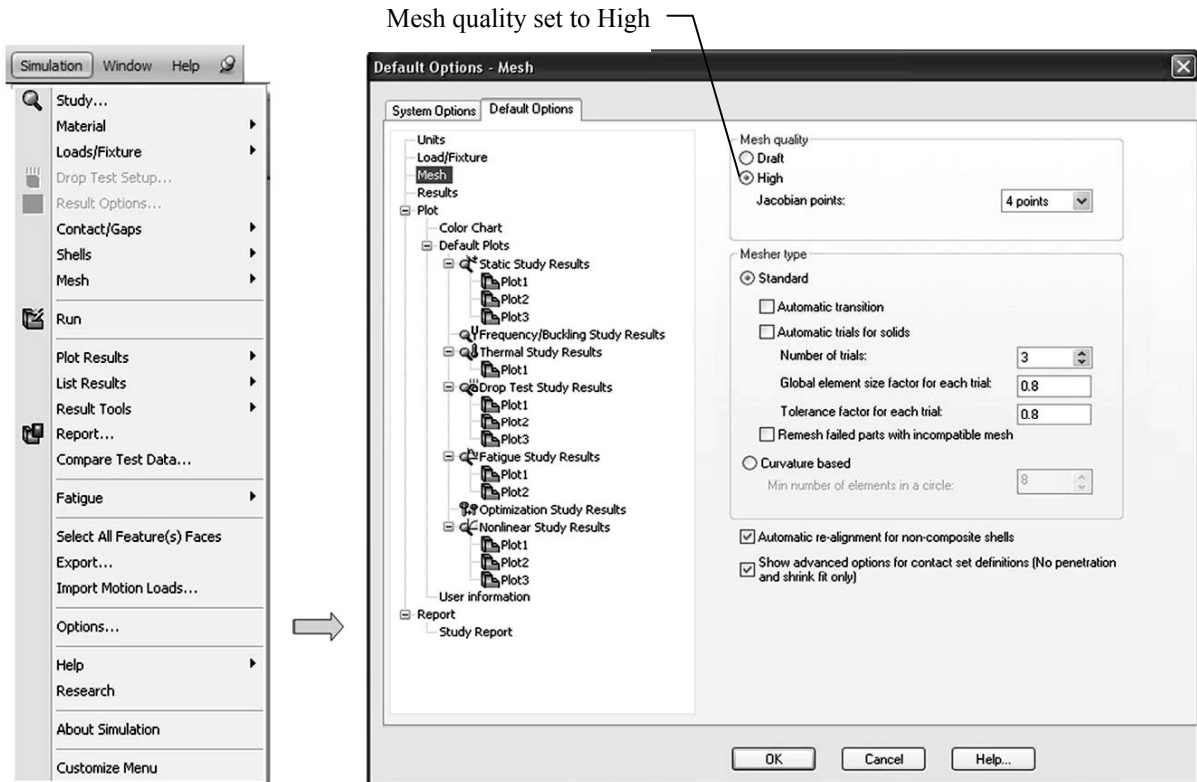


Figure 2-12: Mesh settings in the Options window

Use this window to verify that mesh quality is set to High.

The difference between **High** and **Draft** mesh quality is that:

- ❑ Draft quality mesh uses first order elements
- ❑ High quality mesh uses second order elements

Differences between first and second order elements were discussed in chapter 1.

Now, right-click the *Mesh* folder to display the pop-up menu (Figure 2-13).

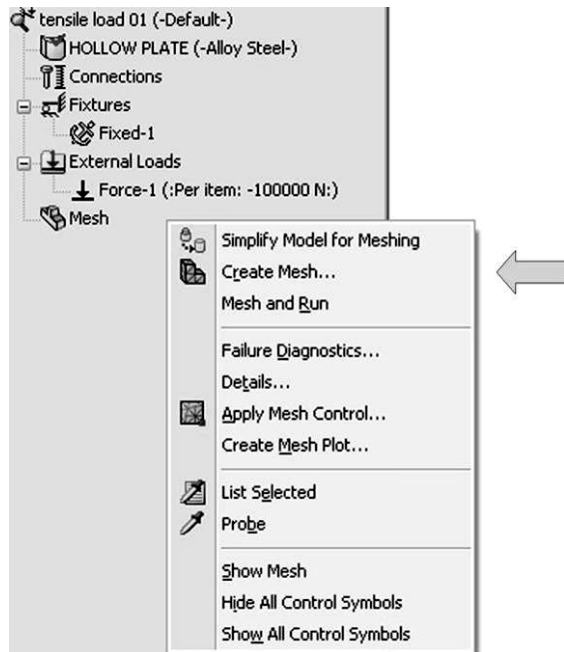


Figure 2-13: Mesh pop-up menu

In the pop-up menu, select **Create Mesh**. This opens the **Mesh** window (Figure 2-14) which offers a choice of element size and element size tolerance.

This exercise reinforces the impact of mesh size on results. Therefore, we will solve the same problem using three different meshes: coarse, medium (default) and fine. Figure 2-14 shows the respective selection of meshing parameters to create the three meshes.

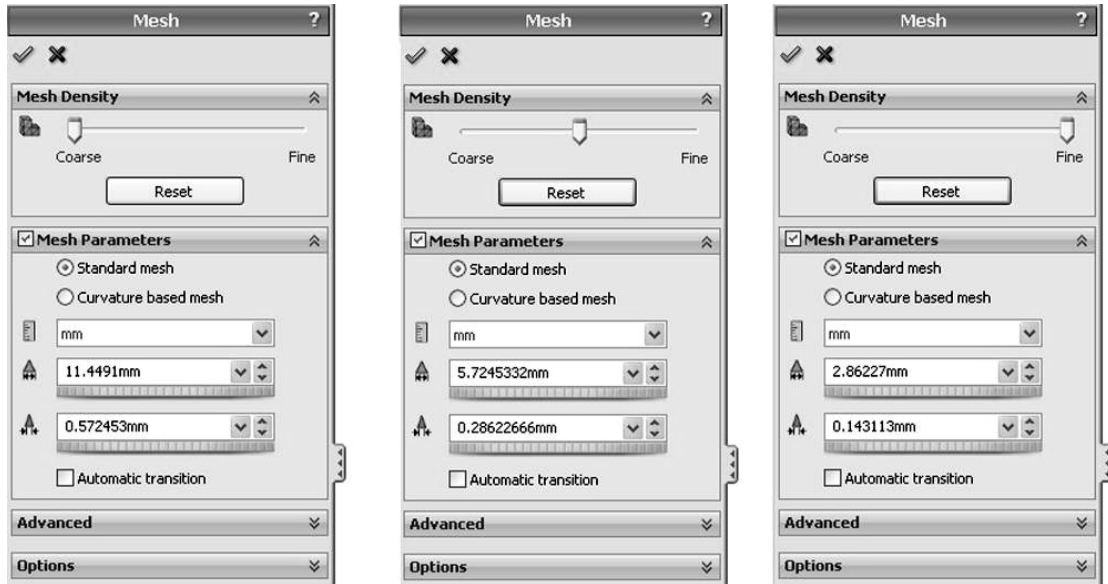


Figure 2-14: Three choices for mesh density from left to right: coarse, medium (default), and fine

Select mesh Parameters to see the element size. In all three cases use Standard mesh. Note different slider positions in the three windows.

The medium mesh density, shown in the middle window in Figure 2-14, is the default that SolidWorks **Simulation** proposes for meshing our model. The element size of 5.72 mm and the element size tolerance of 0.286mm are established automatically based on the geometric features of the SolidWorks model. The 5.72-mm size is the characteristic element size in the mesh, as explained in Figure 2-15. The default tolerance is 5% of the global element size. If the distance between two nodes is smaller than this value, the nodes are merged unless otherwise specified by contact conditions (contact conditions are not present in this model).

Mesh density has a direct impact on the accuracy of results. The smaller the elements, the lower are the discretization errors, but the meshing and solving time both take longer. In the majority of analyses with SolidWorks **Simulation**, the default mesh settings produce meshes that provide acceptable discretization errors, while keeping solution times reasonably short.

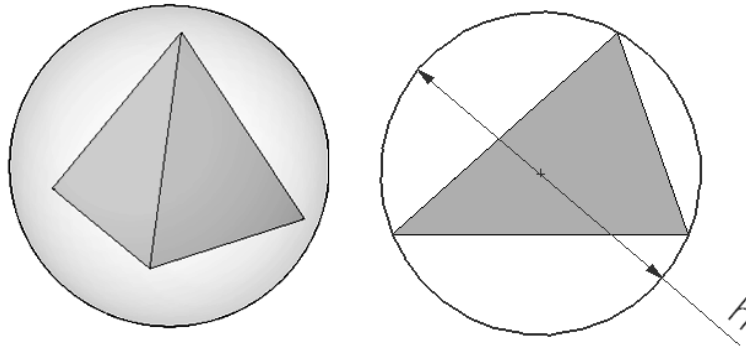


Figure 2-15: Characteristic element size for a tetrahedral element

The characteristic element size of a tetrahedral element is the diameter h of a circumscribed sphere (left). This is easier to illustrate with the 2-D analogy of a circle circumscribed on a triangle (right).

Right-click the *Mesh* folder again and select **Create...** to open the **Mesh** window.

With the **Mesh** window open, set the slider all the way to the left (as illustrated in Figure 2-14, left) to create a coarse mesh, and click the green checkmark button. The mesh will be displayed as shown in Figure 2-16.

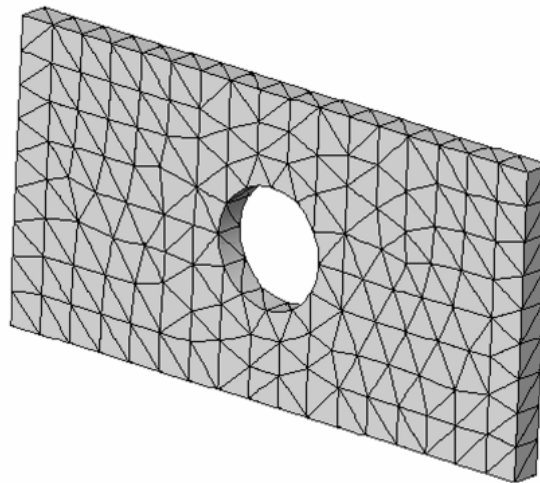


Figure 2-16: Coarse mesh created with second order, solid tetrahedral elements

*You can control the mesh visibility by selecting **Hide Mesh** or **Show Mesh** from the pop-up menu shown in Figure 2-13.*

To start the solution, right-click the *tensile load 01* study folder which displays a pop-up menu (Figure 2-17). Select **Run** to start the solution.

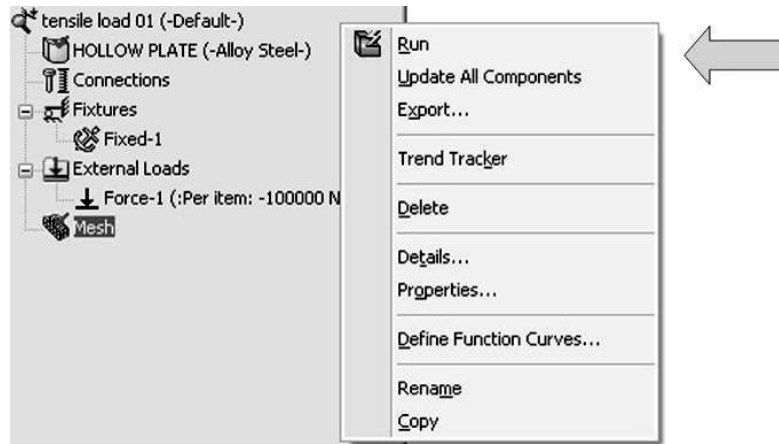


Figure 2-17: Pop-up menu for the *Study* folder

*Start the solution by right-clicking the *tensile load 01* folder to display a pop-up menu. Select **Run** to start the solution.*

The solution can be executed with different properties, which we will investigate in later chapters. You can monitor the solution progress while the solution is running (Figure 2-18).

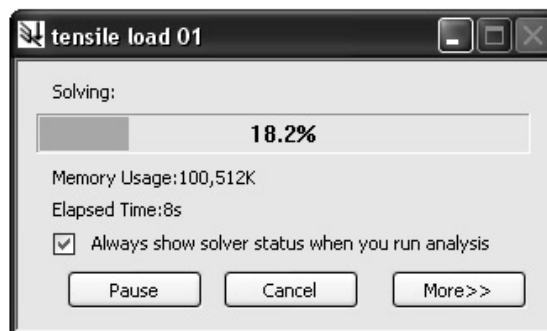


Figure 2-18: Solution Progress window

The Solver reports solution progress while the solution is running.

If the solution fails, the failure is reported as shown in Figure 2-19.

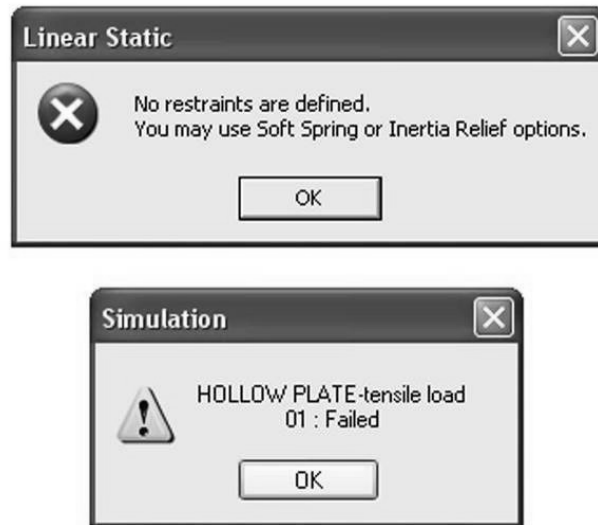


Figure 2-19: Failed solution warning window

Here solution of model with no restraints was attempted. Once the error message has been acknowledged (top), solver displays the final outcome of solution (bottom).

When the solution completes successfully, **Simulation** creates a *Results* folder with result plots which are defined in **Simulation Default Options** as shown in Figure 2-20.

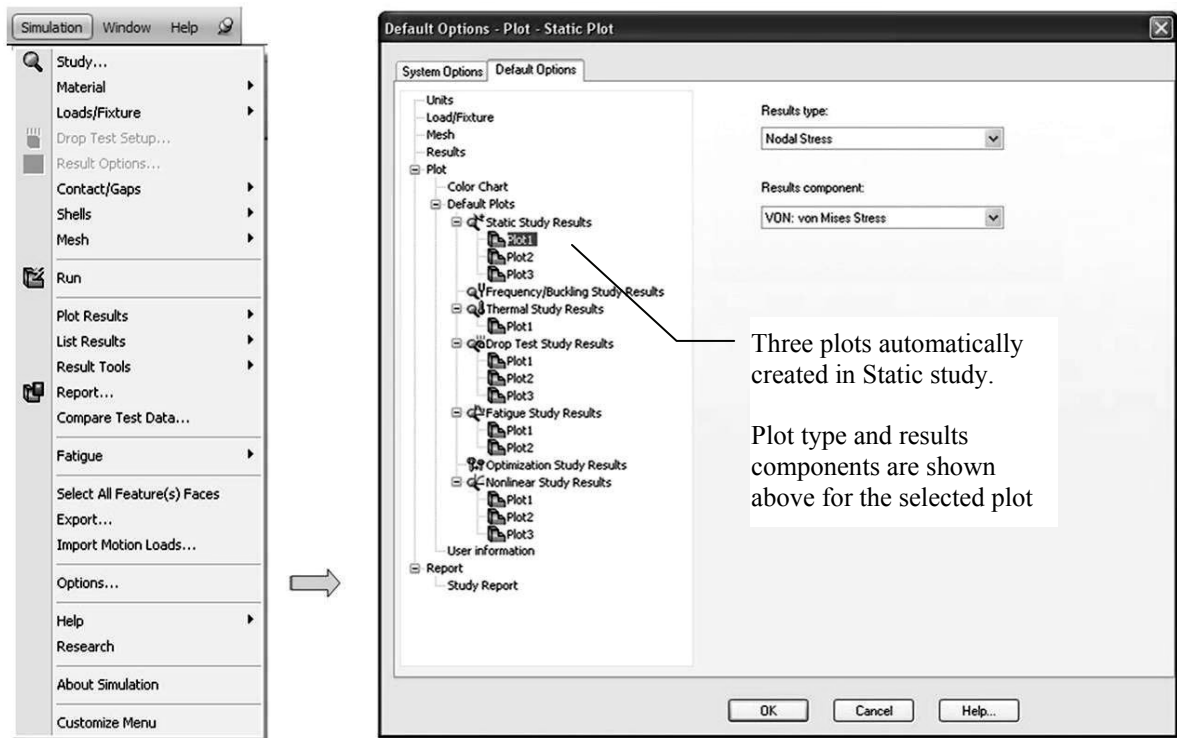


Figure 2-20: Plots that are automatically placed in *Results* folder are defined in Simulation Default Options

Review the definition of all plots in a Static study.

In a typical configuration three plots are created automatically in the Static study:

- ❑ *Stress1* showing von Mises stresses
- ❑ *Displacement1* showing resultant displacements
- ❑ *Strain1* showing equivalent strain

Make sure that the above plots are defined in your configuration, if not, define them.

Once the solution completes, you can add more plots to the *Results* folder. You can also create subfolders in the *Results* folder to group plots (Figure 2-21).

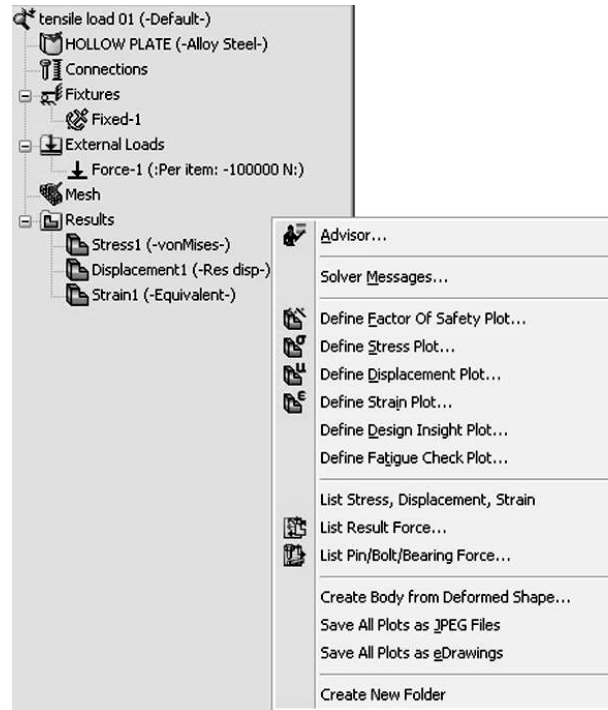


Figure 2-21: More plots and folders can be added to *Results* folder

Right-clicking on Results activates this pop-up menu from which plots may be added to the Results folder.

To display stress results, double-click on the **Stress1** icon in the *Results* folder or right-click it and select **Show** from the pop-up menu. The default stress plot is shown in Figure 2-22.

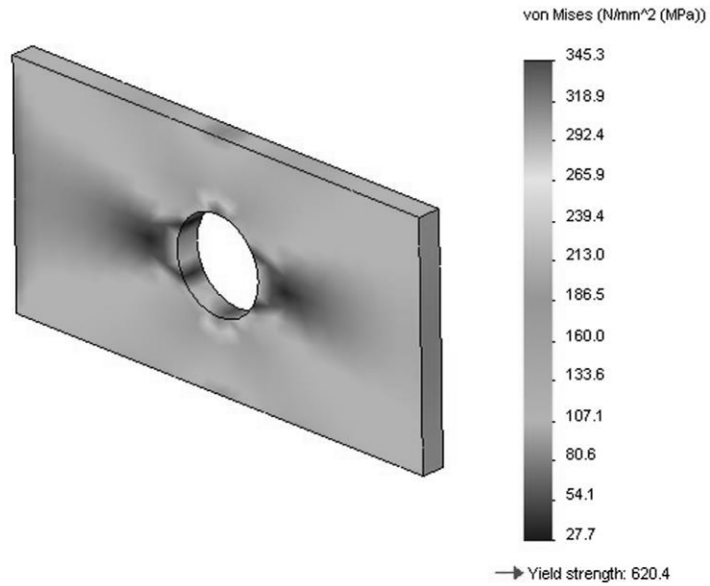


Figure 2-22: Stress plot displayed using default stress plot settings

Von Mises stress results are shown by default in the stress plot window. Notice that results are shown in [MPa] and the highest stress 345 MPa is below the material yield strength 620 MPa. The actual numerical results may differ slightly depending on solver, software version and service pack used.

Once the stress plot is showing, right-click the stress plot icon to display the pop-up menu featuring different plot display options (Figure 2-23).



Figure 2-23: Pop-up menu with plot display options

Any plot can be modified using selections from the pop-up menu (left). Arrows relate selections in the pop-up menu to the invoked windows. Explore all selections offered by these three windows. In particular explore color Options accessible from Chart Options, not shown in the above illustration.

We now examine how to modify the stress plot using the **Settings** window shown in Figure 2-23. In **Settings**, select **Discrete** in **Fringe options** and **Mesh** in **Boundary options** to produce the stress plot shown in Figure 2-24.

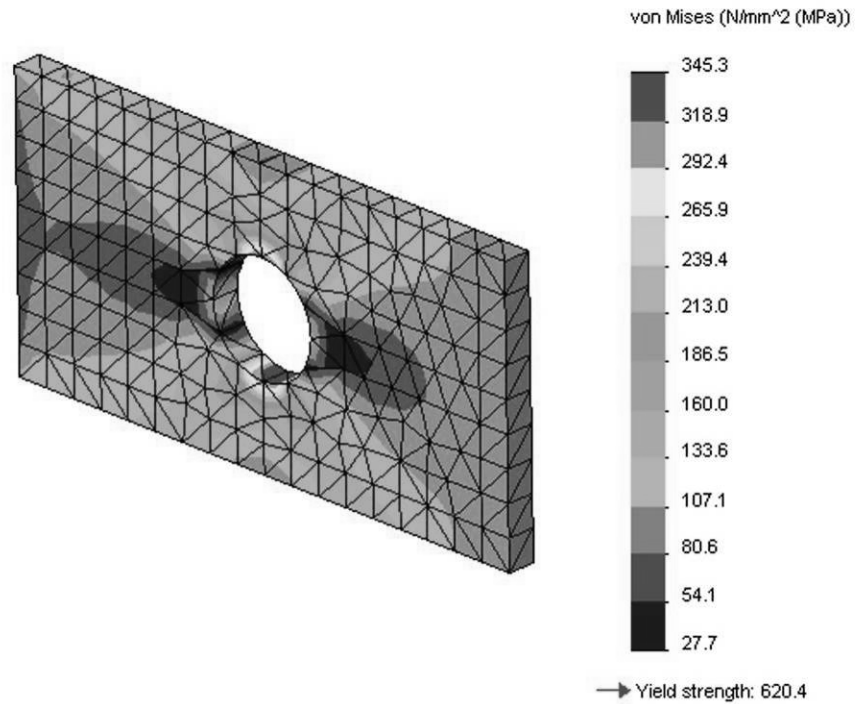


Figure 2-24: The modified stress plot is shown with discrete fringes and the mesh superimposed on the stress plot

The Stress plot in Figure 2-24 shows node values, also called averaged stresses. Element values (or non-averaged stresses) can be displayed by proper selection in the **Stress Plot** window in **Advanced Options**. Node values are most often used to present stress results. See chapter 3 and the glossary of terms in chapter 23 for more information on node values and element values of stress results.

Before you proceed, investigate this stress plot with other selections available in the windows shown in Figure 2-23.

We now review the displacement and strain results. All of these plots are created and modified in the same way. Sample results are shown in Figure 2-25 (displacement) and Figure 2-26 (strain).

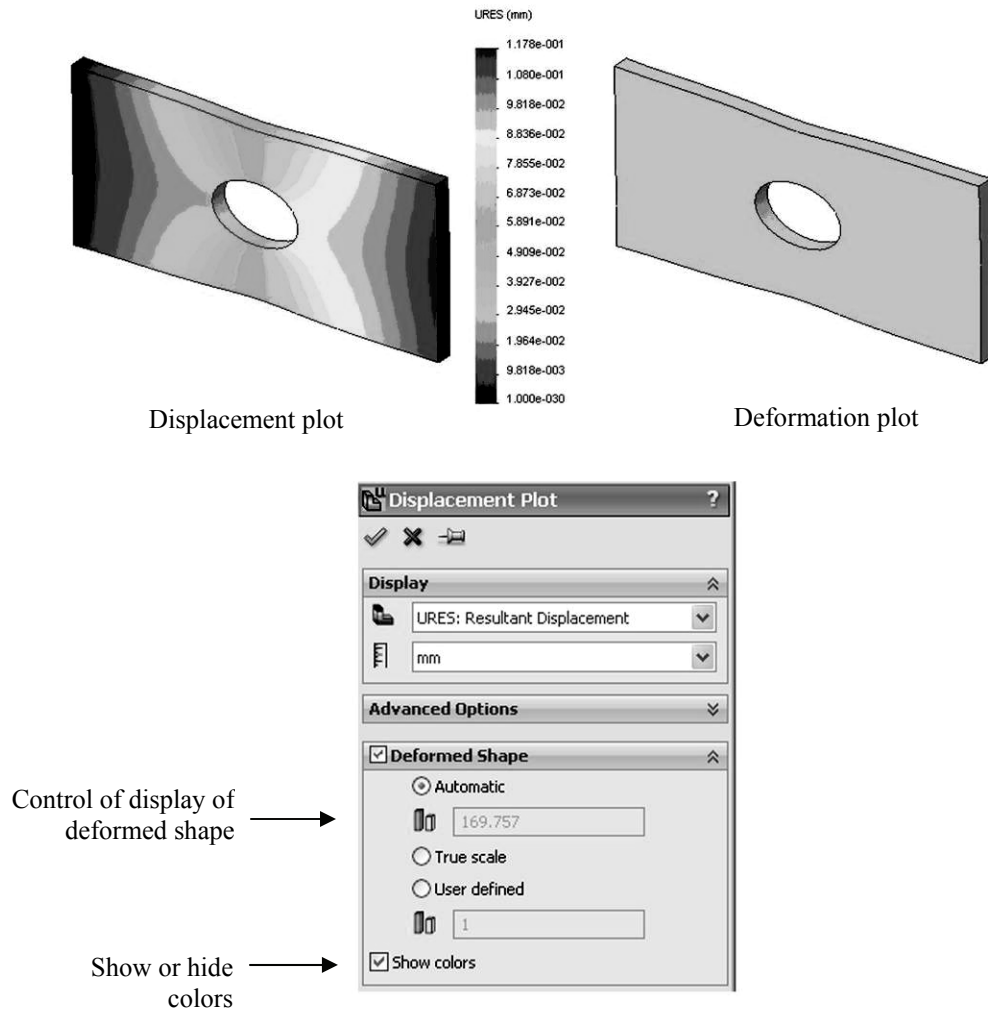


Figure 2-25: Displacement plot (left) and Deformation plot (right)

A Displacement plot can be turned into a Deformation plot by deselecting Show Colors in Displacement Plot in Edit Definition window. The same window has the option of showing the model with an exaggerated scale of deformation as shown above.

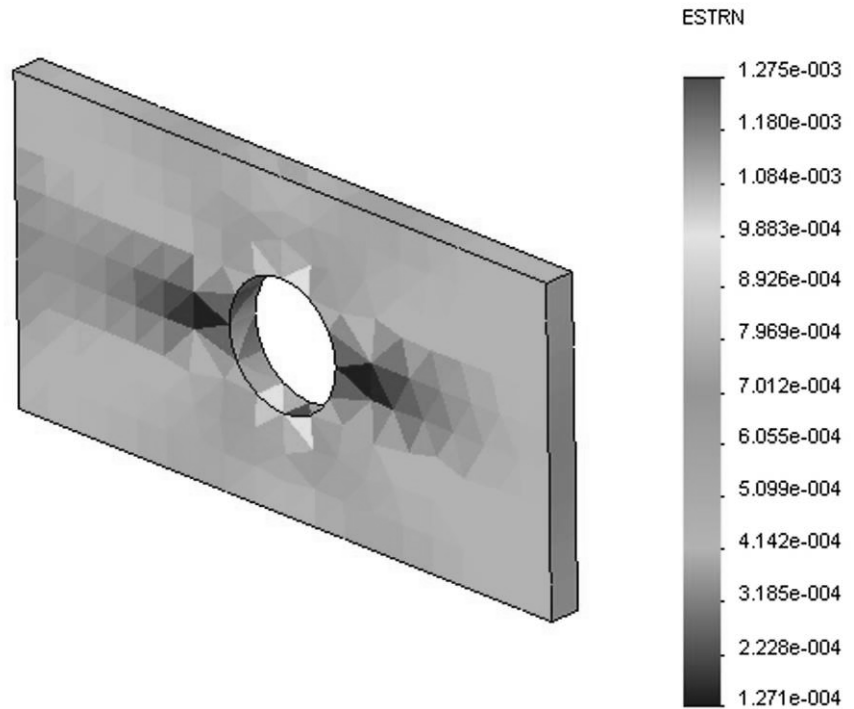


Figure 2-26: Strain results

Strain results are shown here using Element values.

The plots in Figure 2-25 show the deformed shape in an exaggerated scale. You can change the display from deformed to undeformed or modify the scale of deformation in the **Displacement Plot**, **Stress Plot**, and **Strain Plot** windows, activated by right-clicking the plot icon, then selecting **Edit Definition**.

Now, construct a **Factor of Safety** plot using the menu shown in Figure 2-21. The definition of the **Factor of Safety** plot requires three steps. Follow steps 1-3 using the selection shown in Figure 2-27. Review Help to learn about failure criteria and their applicability to different materials.

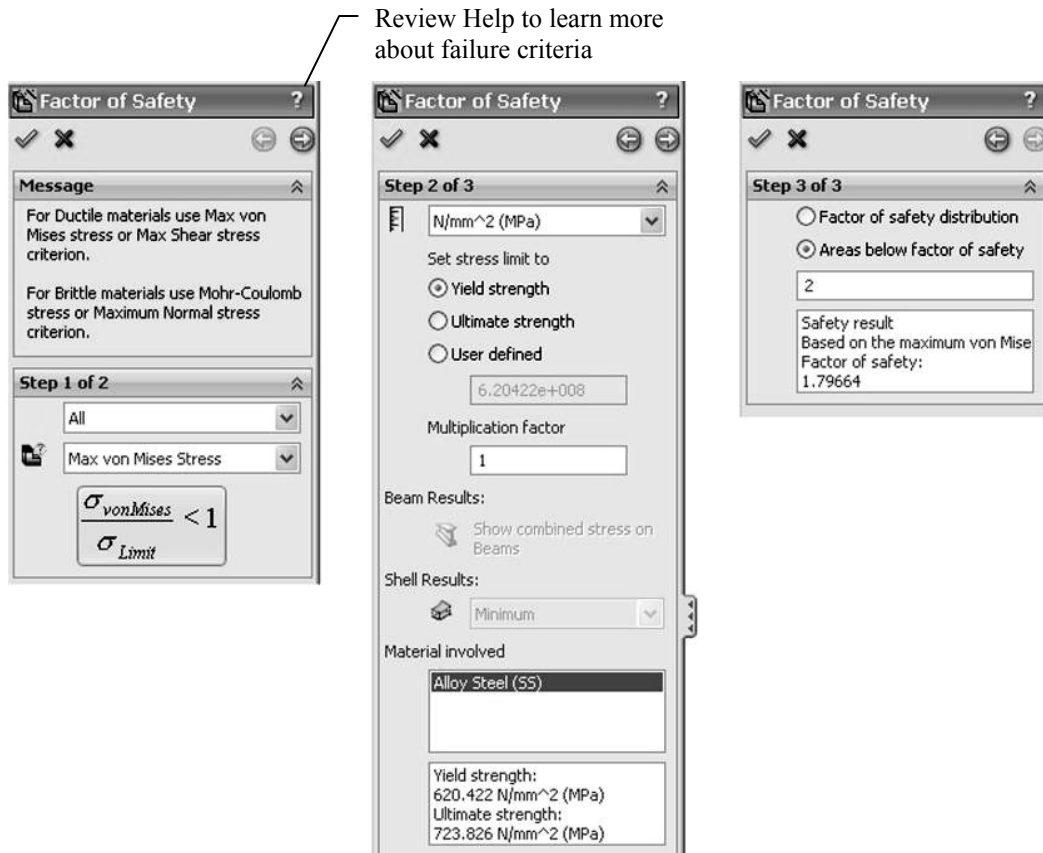


Figure 2-27: Three windows show the three steps in the Factor of Safety plot definition. Select Max von Mises Stress criterion in the first window.

To move through steps, click on the right and left arrows located at the top of the Factor of Safety dialog.

Step 1 selects the failure criterion, step 2 selects display units and sets the stress limit, step 3 selects what will be displayed in the plot. Here we select areas below the factor of safety 2.

The factor of safety plot in Figure 2-28 shows the area where the factor of safety is below the specified.

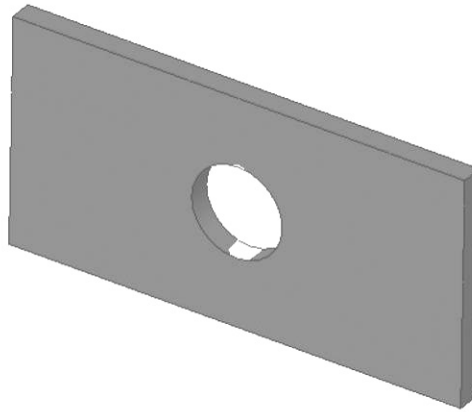


Figure 2-28: Red color (shown as white in this grayscale illustration) displays the areas where the factor of safety falls below 2

We have completed the analysis with a coarse mesh and now wish to see how a change in mesh density will affect the results. Therefore, we will repeat the analysis two more times using medium and fine density meshes respectively. We will use the settings shown in Figure 2-14. All three meshes used in this exercise (coarse, medium, and fine) are shown in Figure 2-29.

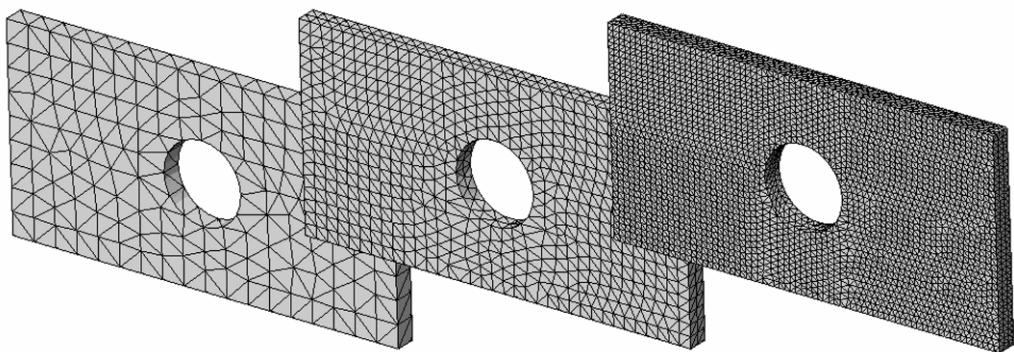


Figure 2-29: Coarse, medium, and fine meshes

Three meshes used to study the effects of mesh density on results.

To compare the results produced by different meshes, we need more information than is available in the plots. Along with the maximum displacement and the maximum von Mises stress, for each study we need to know:

- ❑ The number of nodes in the mesh.
- ❑ The number of elements in the mesh.
- ❑ The number of degrees of freedom in the model.

The information on the number of nodes and number of elements can be found in **Mesh Details** (Figure 2-30).

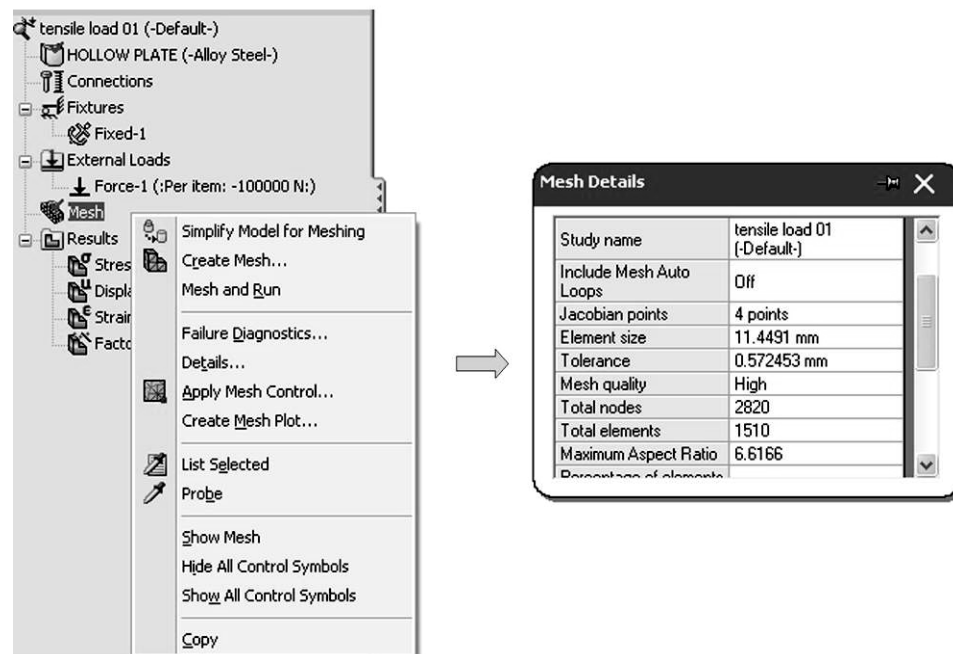


Figure 2-30: Meshing details window

Right-click the Mesh folder and select Details from the pop-up menu to display the Mesh Details window. Note that information on the number of degrees of freedom is not available here.

The most convenient way to find the number of nodes, elements and degrees of freedom is to use the pop-up menu activated by right-clicking on the *Results* folder (Figure 2-31).

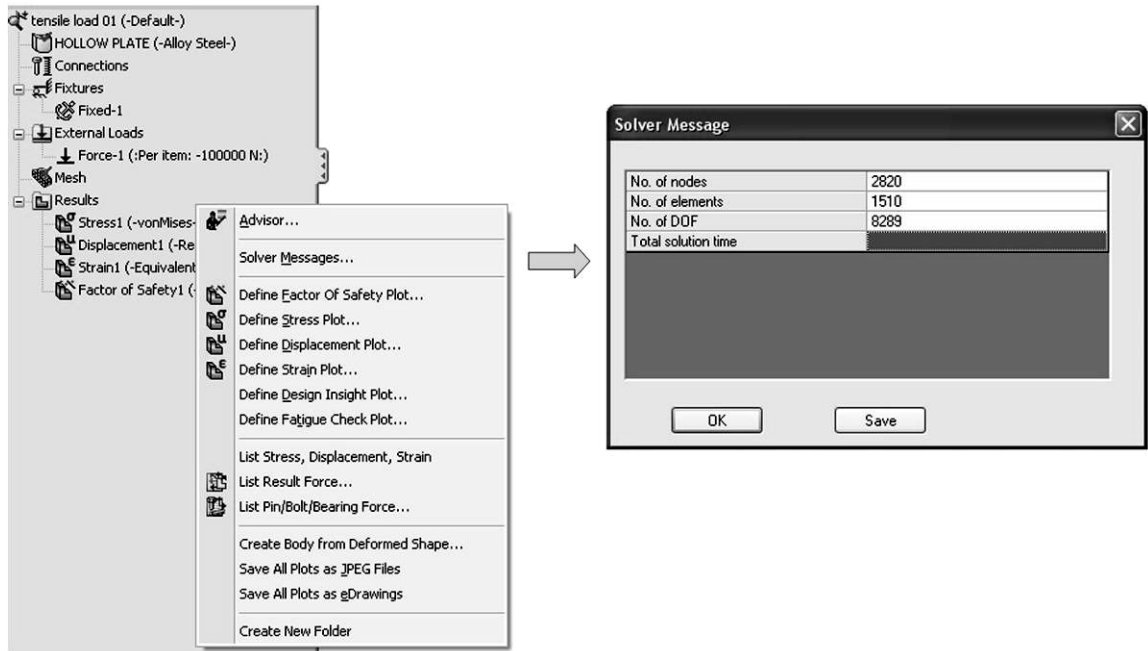


Figure 2-31: The Solver Message window lists information pertaining to the solved study

Right-click on the Results folder and select Solver Messages from the pop-up menu to display the number of nodes, elements and degrees of freedom.

Now create and run two more studies: *tensile load 02* with default element size and *tensile load 03* with fine element size, as shown in Figure 2-14. To create a new study we could just repeat the same steps as before but an easier way is to copy a study. To copy a study, follow the steps in Figure 2-32.

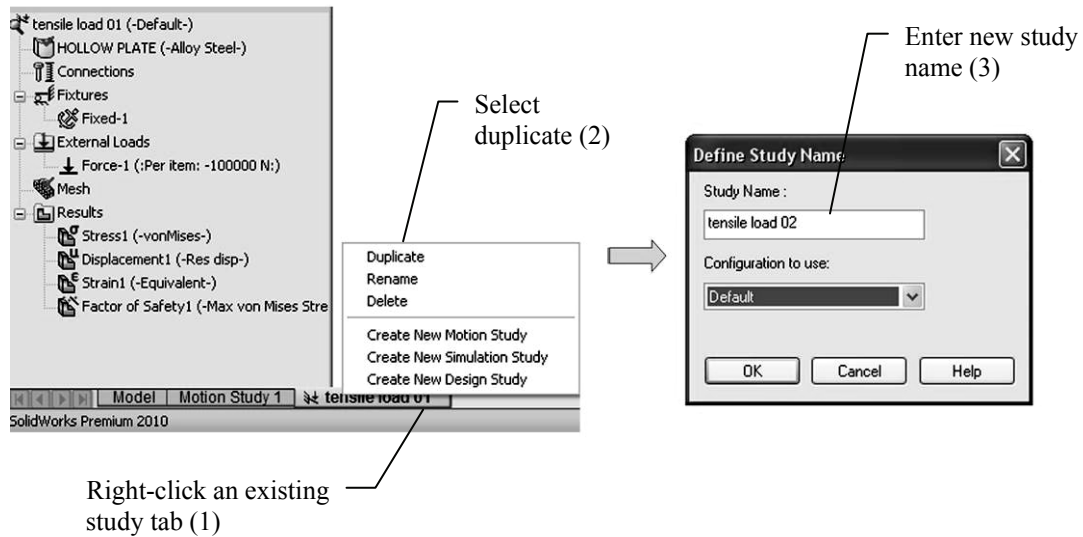


Figure 2-32: A study can be copied into another study in three steps as shown

Note that all definitions in a study (material, restraints, loads, mesh) can also be copied individually from one study to another by dragging and dropping them into new study tab.

A study is copied complete with results and plot definitions. Before remeshing, study *tensile load 02* with the default element size mesh, you must acknowledge the warning message shown in Figure 2-33.

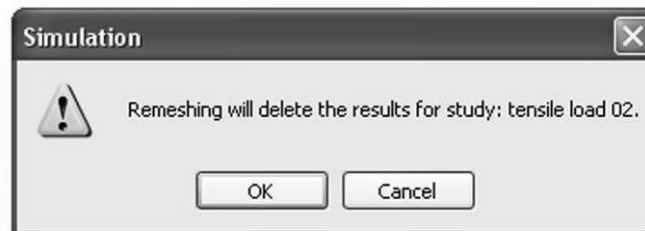


Figure 2-33: Remeshing deletes any existing results in the study

The summary of results produced by the three studies is shown in Figure 2-34.

Study	Element size [mm]	Max. resultant displacement [mm]	Max. von Mises stress [MPa]	Number of elements	Number of nodes	Number of DOF
<i>tensile load 01</i>	11.45	0.1178	345	2820	1510	8289
<i>tensile load 02</i>	5.72	0.1180	372	12204	7024	36057
<i>tensile load 03</i>	2.86	0.1181	378	84427	55222	251796

Figure 2-34: Summary of results produced by the three meshes

Note that these results are based on the same problem. Differences in the results arise from the different mesh densities used in studies tensile load 01, tensile load 02, and tensile load 03.

The actual numbers in this table may vary slightly depending on type of solver and release of software used for solution.

Figure 2-35 shows the maximum resultant displacement and the maximum von Mises stress as function of the number of degrees of freedom. The number of degrees of freedom is in turn a function of mesh density.

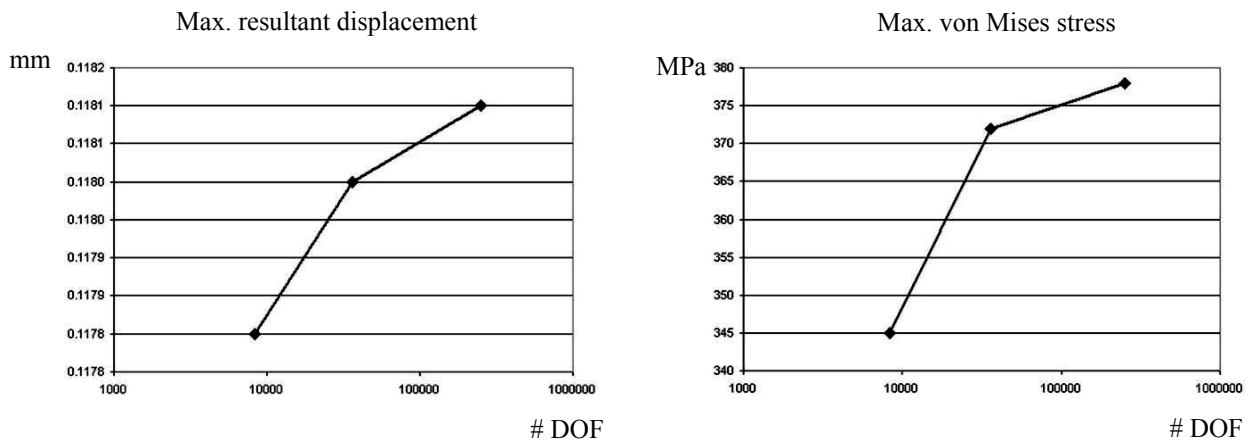


Figure 2-35: Maximum resultant displacement (left) and maximum von Mises stress (right)

Both are plotted as a function of the number of degrees of freedom in the model. The three points on the curves correspond to the three models solved. Straight lines connect the three points only to visually enhance the graphs.

Having noticed that the maximum displacement increases with mesh refinement, we can conclude that the model becomes “softer” when smaller elements are used. With mesh refinement, a larger number of elements allows for better approximation of the real displacement and stress field. Therefore, we can say that the artificial constraints imposed by element definition become less imposing with mesh refinement.

Displacements are the primary unknowns in structural FEA, and stresses are calculated based on displacement results. Therefore, stresses also increase with mesh refinement. If we continued with mesh refinement, we would see that both the displacement and stress results converge to a finite value which is the solution of the mathematical model. Differences between the solution of the FEA model and the mathematical model are due to discretization errors, which diminish with mesh refinement.

We will now repeat our analysis of the HOLLOW PLATE by using prescribed displacements in place of a load. Rather than loading it with a 100000N force that has caused a 0.118 mm displacement of the loaded face, we will apply a prescribed displacement of 0.118 mm to this face to see what stresses this causes. For this exercise, we will use only one mesh with default (medium) mesh density.

Define the fourth study, called *prescribed displ.* The easiest way to do this is to copy study *tensile load 02*. The definition of material properties, the fixed restraint to the left-side end-face and mesh are all identical to the previous design study. We need to delete the load (right-click the load icon and select **Delete**) and apply in its place the prescribed displacement.

To apply the prescribed displacement to the right-side end-face, create a new **Fixed Geometry** by right-clicking the **Fixtures** folder and select **Advanced Fixtures** from the pop-up menu. This opens the **Fixture** definition window. Select **On Flat Face** from the Advanced menu and define displacement as shown in Figure 2-36. Check **Reverse direction** to obtain displacement in the tensile direction. Note that the direction of a prescribed displacement is indicated by a restraint symbol.

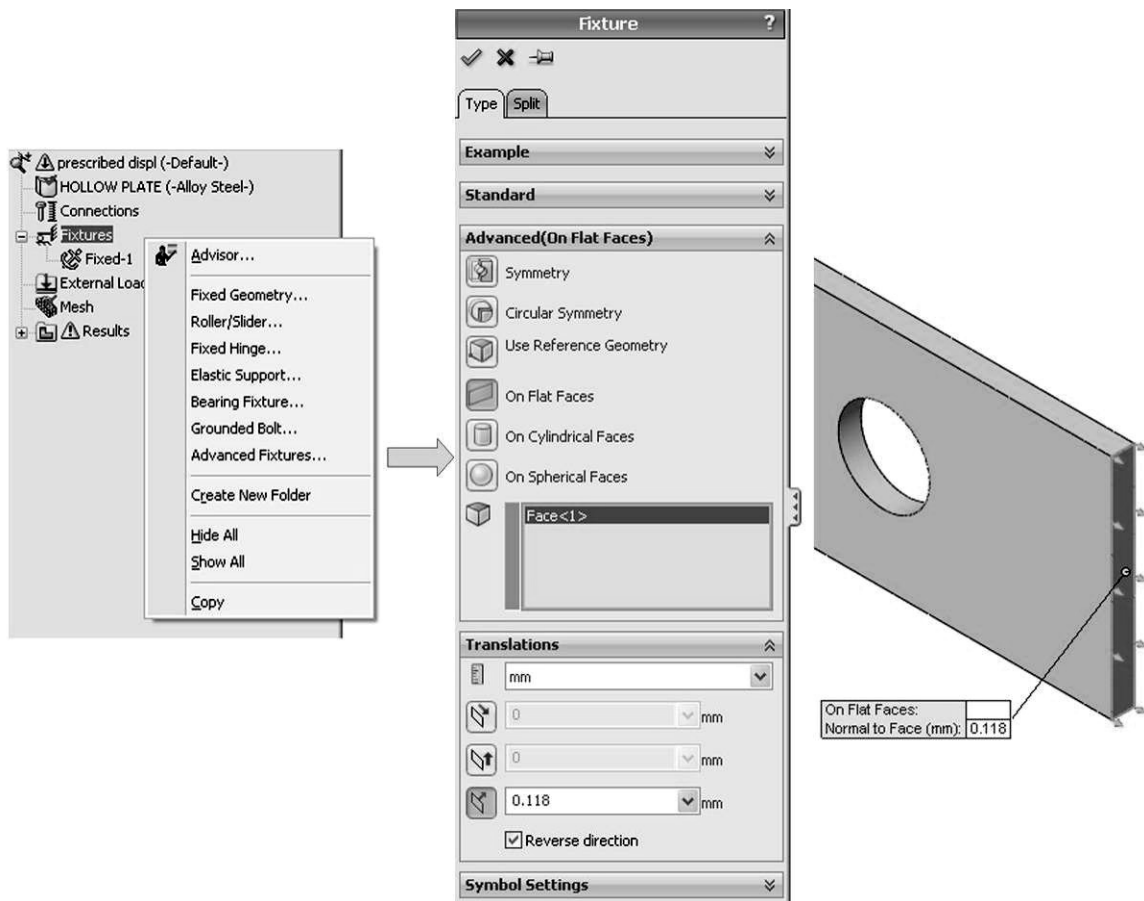


Figure 2-36: Restraint definition window

The prescribed displacement of 0.118 mm is applied to the same face where the tensile load of 100000N had been applied. The size and color of this symbol can be changed using Symbol Settings. The color and size of all load and restraint symbols is controlled the same way.

The visibility of all load and restraints symbols is controlled by right-clicking the symbol and making the desired choice (**Hide/Show**). All load symbols and all restraint (fixture) symbols may also be turned on/off all at once by right-clicking **Fixtures** or **External loads** folders and selecting **Hide all/Show all** from the pop-up menu.

Once prescribed displacement is defined to the end face, it overrides any previously applied loads to the same end face. While it is better to delete the load in order to keep the model clean, the load has no effect if a prescribed displacement is applied to the same entity and in the same direction.

Figures 2-37 compares stress results for the model loaded with force to the model loaded with prescribed displacement.

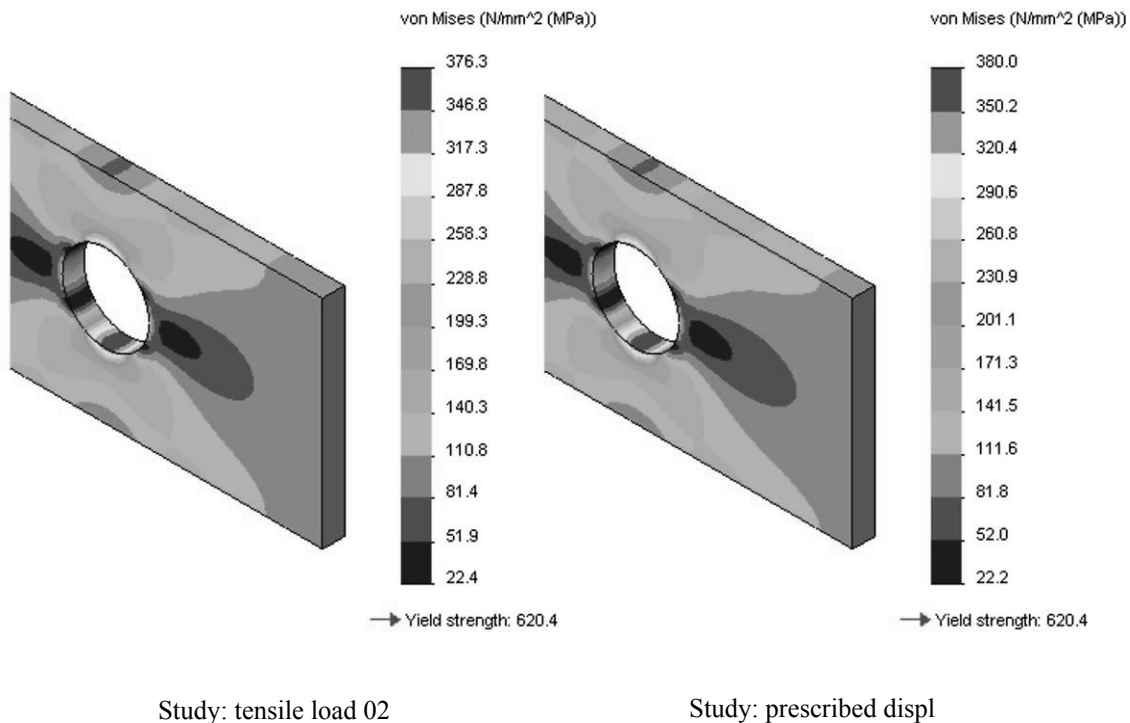


Figure 2-37: Comparison of von Mises stress results

Von Mises stress results with load applied as force are displayed on the left and Von Mises stress results with load applied as prescribed displacement are displayed on the right.

Results produced by applying a force load and by applying a prescribed displacement load are very similar, but not identical. The reason for this discrepancy is that in the model loaded by force, the loaded face is allowed to deform. In the prescribed displacement model, this face remains flat, even though it experiences displacement as a whole. Also, while the prescribed displacement of 0.118 mm applies to the entire face in the prescribed displacement model, it is only seen as a maximum displacement in one point in the force load model. You may plot displacement along the edge of the end face by following the steps in Figure 2-38.

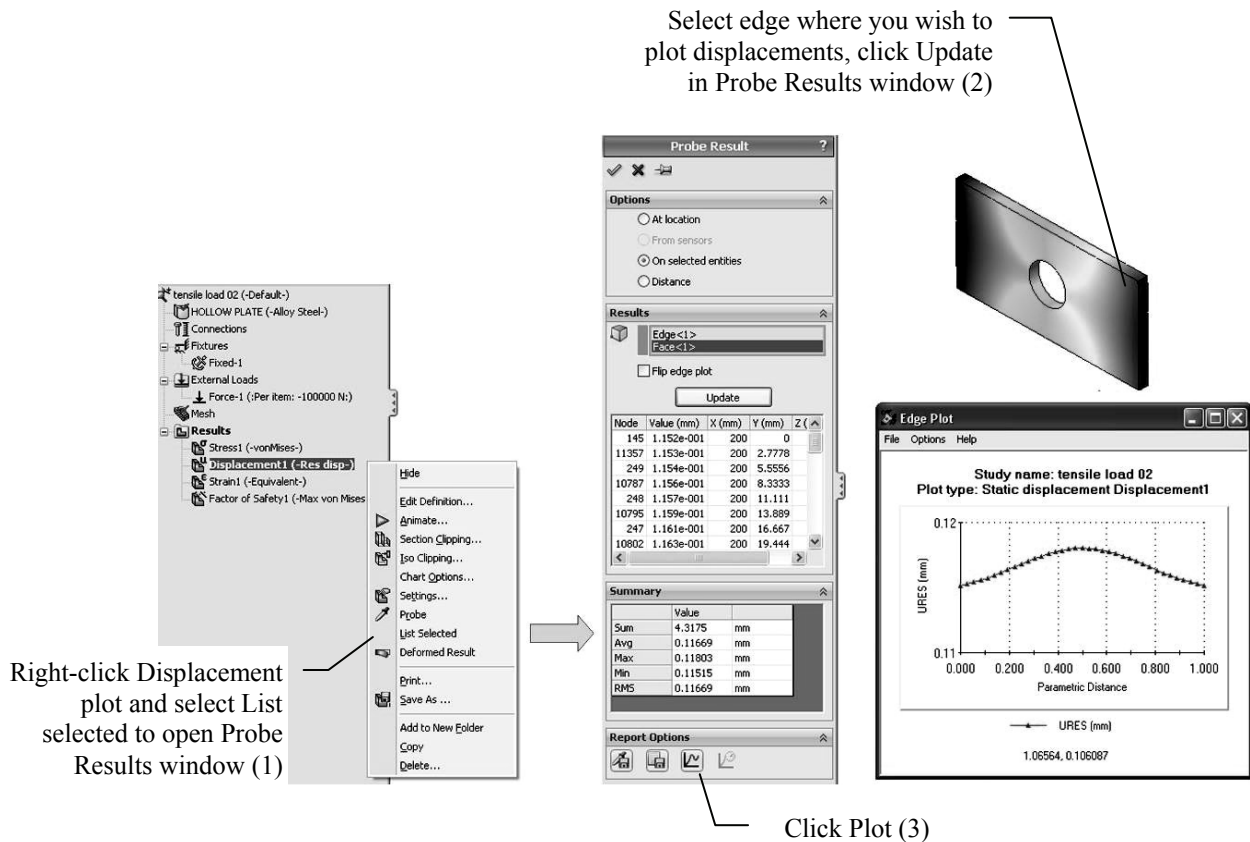


Figure 2-38: Plotting displacement along the edge of force loaded face in study *tensile load 02*

Follow steps 1, 2, and 3 to produce a graph of displacements along the loaded edge. Repeat this exercise for a model loaded with prescribed displacement to verify that displacement is constant along the edge.

We conclude the analysis of the HOLLOW PLATE by examining the reaction forces using the results of study *tensile load 02*. In the study *tensile load 02*, right-click *Results*. From the pop-up menu, select **List Result Force** to open the **Result Force** window. Select the face where the fixed restraint is applied and click the **Update** button. Information on reaction forces will be displayed as shown in Figure 2-39.

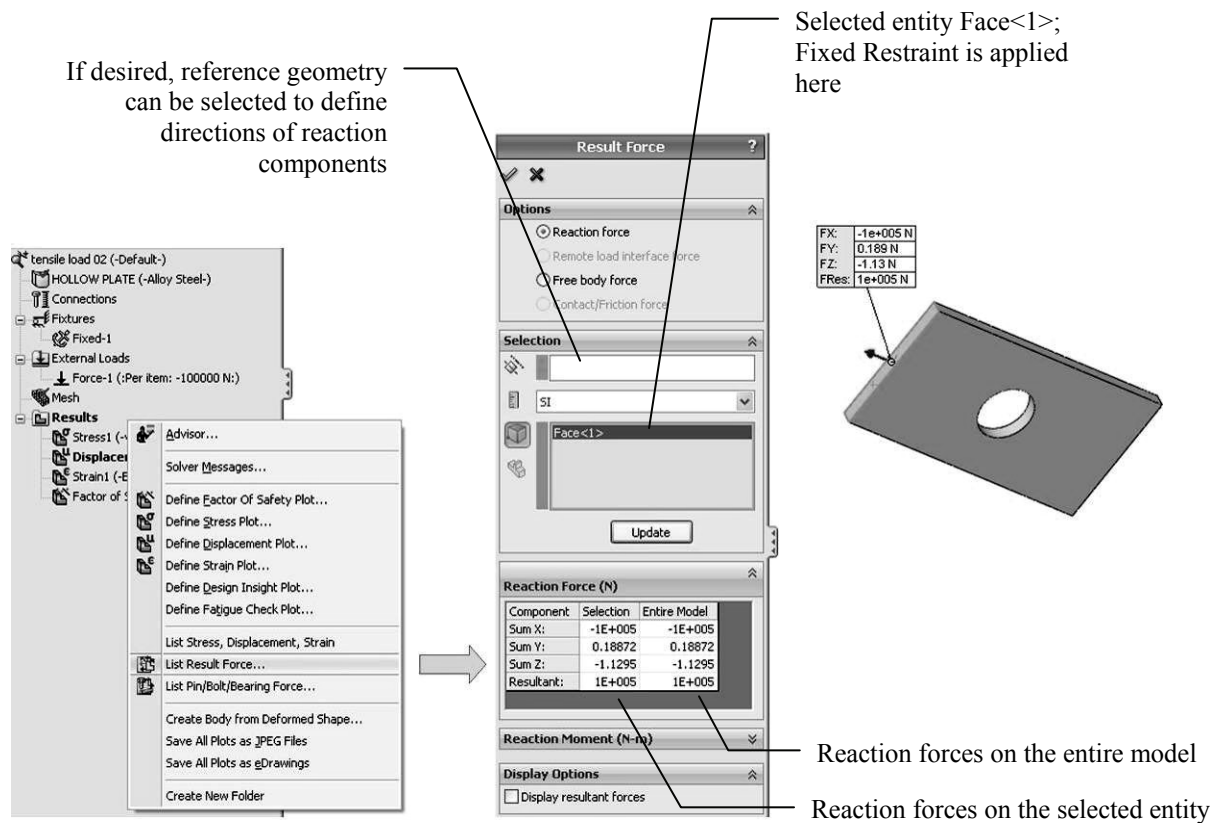


Figure 2-39: Result Force window

Reaction forces can also be displayed in components other than those defined by the global reference system. To do this, reference geometry such as plane or axis must be selected.

A note on where **Simulation** results are stored: All studies are saved with the SolidWorks part or assembly model. Mesh data and results of each study are stored separately in *.CWR files. For example, the mesh and results of study *tensile load 02* have been stored in the file:

HOLLOW PLATE-tensile load 02.CWR.

When the study is opened, the CWR file is unzipped into a number of different files depending on the type of study. Upon exiting SolidWorks **Simulation** (which is done by means of deselecting SolidWorks **Simulation** from the list of add-ins, or by closing the SolidWorks model), all files are compressed back allowing for convenient backup of SolidWorks **Simulation** results.

The location of CWR files is specified in the **Default Options** window which is called from Simulation main menu first shown in Figure 2-4. For easy reference, the **Default Options** window is shown again in Figure 2-40).

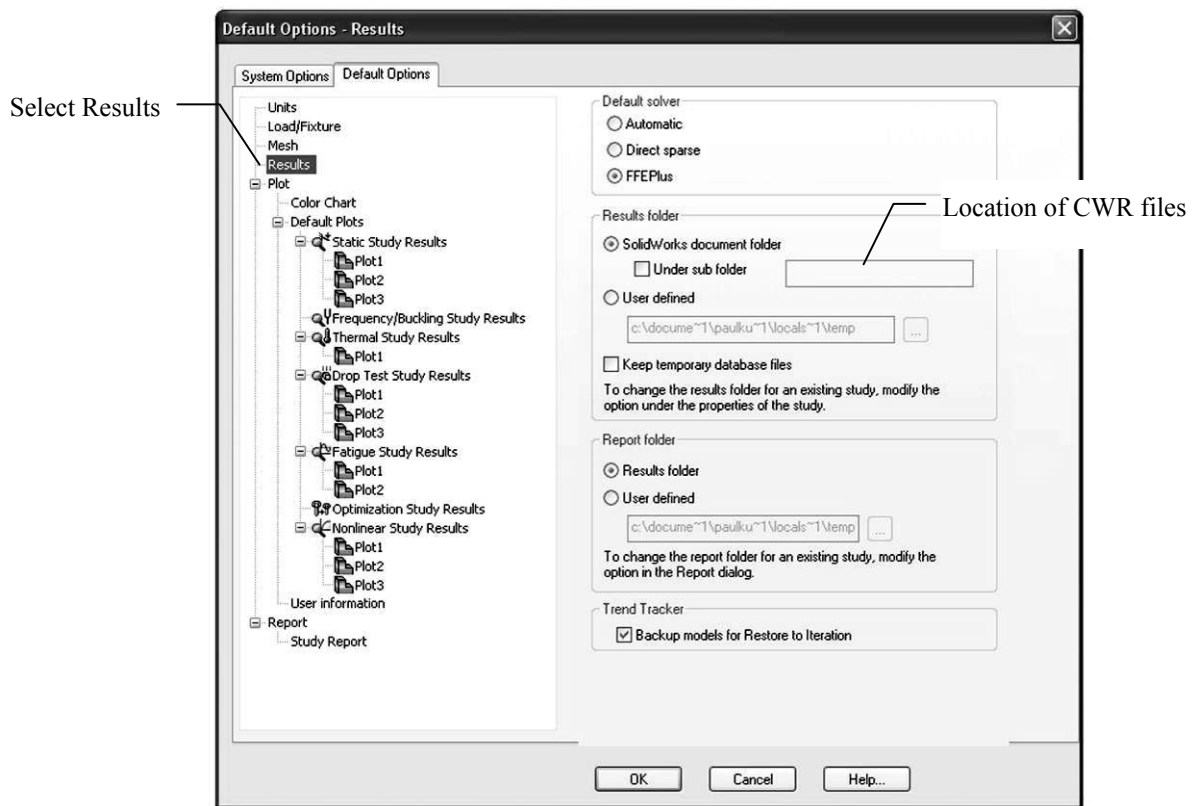


Figure 2-40: Location of solution database files

Using these settings, CWR files are located in SolidWorks document folder.